Engine, Pump and Compressor Power Take-Off Clutches





Engine, Pump and Compressor Clutches

Contents

Application Examples 1 Selection 2 Types of Shafts and Drives 4 Electrical Ratings 5 Clutch Mounting 5 Tapered Bore Clutches: 5 Flange Mount, Single Row Bearing 6 Flange or Yoke Direct Drive, Double Row Bearing 7 Flange Mount, Double Row Bearing 8-13 Special Construction, Flange Mount, Double Row Bearing 14 Straight Bore Clutches: 15-17 Bearing Mount 15-17 Special Construction 18-21 Shaft Mount 22-23	
Types of Shafts and Drives. 4 Electrical Ratings. 5 Clutch Mounting. 5 Tapered Bore Clutches: 5 Flange Mount, Single Row Bearing	Application Examples 1
Electrical Ratings 5 Clutch Mounting 5 Tapered Bore Clutches: 5 Flange Mount, Single Row Bearing 6 Flange or Yoke Direct Drive, Double Row Bearing 7 Flange Mount, Double Row Bearing 8-13 Special Construction, Flange Mount, Double Row Bearing 14 Straight Bore Clutches: 15-17 Special Construction 18-21	Selection
Clutch Mounting	Types of Shafts and Drives 4
Tapered Bore Clutches: Flange Mount, Single Row Bearing Flange or Yoke Direct Drive, Double Row Bearing 7 Flange Mount, Double Row Bearing 8-13 Special Construction, Flange Mount, Double Row Bearing 14 Straight Bore Clutches: Bearing Mount 15-17 Special Construction	Electrical Ratings5
Flange Mount, Single Row Bearing	Clutch Mounting5
Flange or Yoke Direct Drive, Double Row Bearing	Tapered Bore Clutches:
Flange Mount, Double Row Bearing	Flange Mount, Single Row Bearing6
Special Construction, Flange Mount, Double Row Bearing	Flange or Yoke Direct Drive, Double Row Bearing7
Straight Bore Clutches: Bearing Mount	Flange Mount, Double Row Bearing8-13
Bearing Mount	Special Construction, Flange Mount, Double Row Bearing 14
Special Construction	Straight Bore Clutches:
·	Bearing Mount
Shaft Mount	Special Construction
	Shaft Mount

Glossary	24
Technical Considerations	25
Part Number Index	26
Mobile Power Application Data Form	28

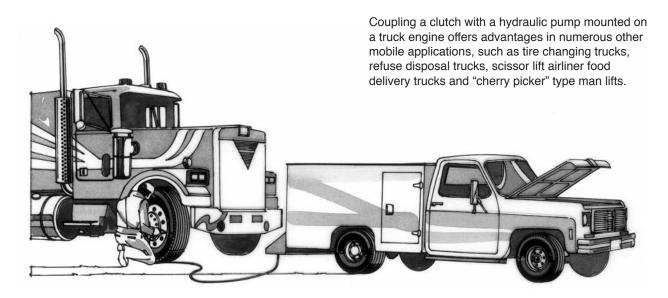
Principles of Operation

An electric clutch operates on the basic principle of magnetic attraction. Closing a switch sends electric current to an electromagnet or "field", producing a strong magnetic attraction which concentrates around the magnetic poles of the field. The magnetic attraction jumps the small air gap between the field and the rotor, effectively making the rotor a spinning magnet. This "magnet" attracts the armature, grips it tightly and causes it to turn with the rotor. Opening the switch turns off the magnet, disengaging the clutch.

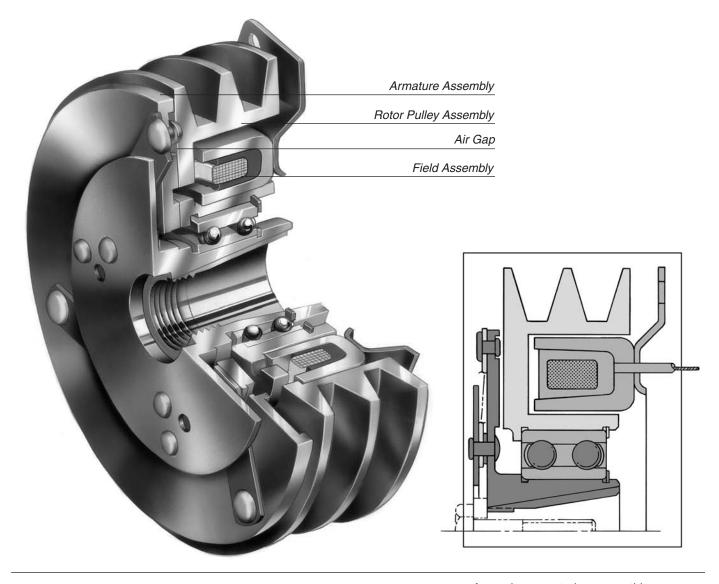
Magnetic Circuit

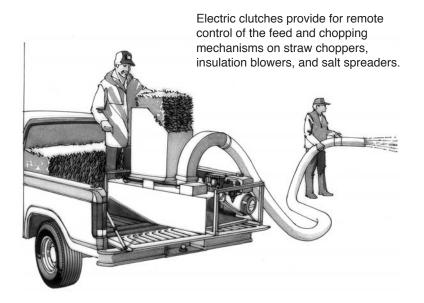
The heart of an electric clutch is its field assembly, with an electric coil to produce magnetism and iron magnet housing to direct that magnetism through the rotor to most effectively attract the armature. The illustration on the right shows how the rotor and armature surfaces of Warner Electric clutches are slotted and grooved. Magnetism, also called magnetic "flux", concentrates at the poles of the field housing and jumps to the outer rims of the rotor. The flux passes through areas where contact is most intimate between the armature and rotor faces. Concentrating the magnetic attraction at these poles rather than over the entire face creates a strong attraction between the armature and rotor. The result is a great deal of torque in a small clutch.

Application Examples

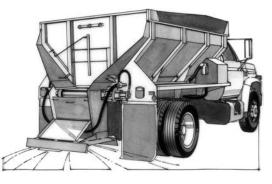


Engine, Pump and Compressor Clutches—Selection





An engine mounted pump on this highway maintenance truck supplies hydraulic pressure for the sand spreader. Mounting an electric clutch on the pump will increase equipment life, improve machine efficiency, and allow the operator to engage and disengage the hydraulic system on demand.



The right clutch for your application

Several specifications must be determined to select the right Warner Electric clutch for a particular application. Compile these from the selection factors discussed on the next four pages and compare them to the clutch specifications found on pages 6 through 23 to find the correct clutch for your application.

Clutch Dimensions

Overall or envelope dimensions of a clutch being considered for an application must be noted to ensure that the clutch will fit into the space allotted.

Choosing the correct size clutch

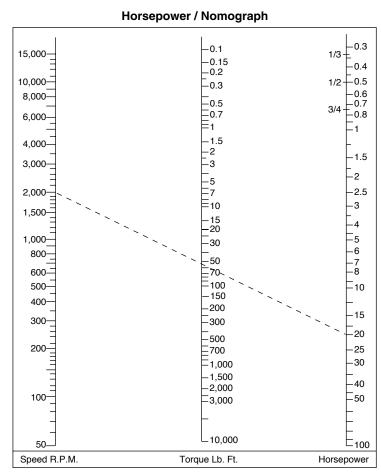
Torque and horsepower calculations are important in choosing the right clutch for your application.

Two important factors needed to determine the static torque required for an application are the driven machine's horsepower and clutch operating speed. Locate horsepower and clutch RPM on their respective columns in the accompanying chart. Draw a straight line between the two points. The torque is the numerical value read from where this line intersects the TORQUE column. This number must be multiplied by a known service factor "K" from the data below to obtain the correct torque requirement for the clutch.

Service Factor "K"

The power pulses of a gasoline or diesel engine result in momentary torque output which is several times higher than the engine's rated torque. Many electric motors can also deliver up to three times their rated output for a short period. A clutch coupled to these power systems must be able to transmit the required torque for these short periods without slipping. Using the chart at right, estimate the "K" service factor which is appropriate for your application.

Torque Determination Chart



Type of Application	"K" Range
Conveyor and augers where a static load must be started by the dynamic torque of the clutch.	K = 3 to 5
Hydraulic pumps where the clutch may have to work against pressure at time of engagement	K = 2.5 to 5
Gasoline or diesel engines where the clutch should be able to stall the engine.	K = 2 to 5
Air compressors	K = 2 to 4
Electric motors where the clutch should be able to stall the motor. Use the overload factor from the motor catalog or use	K = 2 to 3
Light machines where the load is applied after the	
clutch is engaged. (e.g. a lathe)	K = 1.5 to 2.5

Engine, Pump and Compressor Clutches—Selection

Example

A 10 horsepower pump driven by an electric motor operates at 1500 RPM. Using 3 as the value of the "K" factor, determine the required clutch torque capacity.

10 HP at 1500 RPM = 35 lb.ft. torque "K" factor 3 x 40 lb.ft. = 105 lb.ft. torque.

This application will require a clutch with a static torque rating of 105 lb.ft.

Static torgue can also be calculated by using the following formula:

T = 5250 x HP x K **RPM**

Example

A truck mounted hydraulic pump requires 8 horsepower to operate at 2000 RPM. What is the required clutch torque rating?

Using a service factor of 5:

$$T = \frac{5250 \times 8 \times 5}{2000}$$

T = 105 lb.ft.

A clutch with a static torque capacity of 105 lb.ft. is required for this application.

Fluid Power Formulae

T = HP x 5250 x K

RPM

Where:

т –

- T= Torque (lb.ft.)
- HP = Horsepower
- RPM = Speed of Clutch (revolutions per minute)
- CIR = Cubic inch per revolution (hydraulic pump)
- PSI = Pounds per square inch
- "K" = Service factor
 - (see chart on page 2)
- If HP is unknown:

$$HP = GPM \times PSI \times .000583$$

Pump efficiency

Where:

- GPM = Fluid flow in gallons per minute
- PSI = Pressure in pounds per square inch

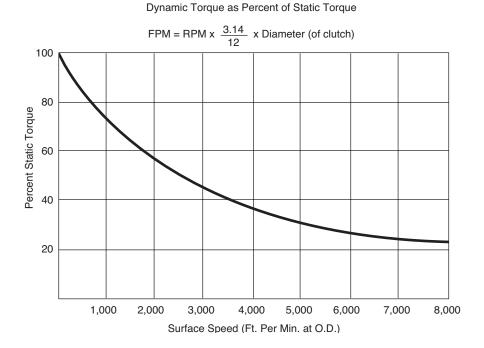
Pump efficiency = normally 85%

Rule of Thumb:

1 HP per gallon @ 1500psi .7 HP per gallon @ 1000psi

If PSI is unknown:

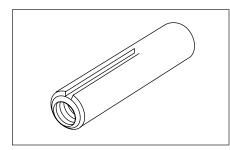
1 cubic inch per revolution equals 16 lb.in. of torque per 100 psi. 1 gallon equals 231 cubic inches.



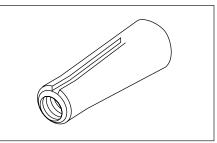
TYPICAL DYNAMIC TORQUE

Shafts

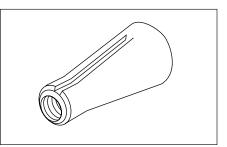
Warner Electric standard clutches are available to adapt to three different shaft configurations:



Straight bore - for through shaft or end of shaft mounting.



Tapered bore (8:1) - shaft tapers $1^{1}/_{2}$ inches per foot, for end of shaft mounting.



Tapered bore (4:1) - shaft tapers three inches per foot, for end of shaft mounting.

Types of Drives

Warner Electric offers clutches which are compatible with the three basic types of drives.

In direct drive arrangements, a drive

engine crankshaft or transmission PTO on one end and the pump or compres-

mounts on the pump or compressor. The "yoke drive" clutches in this section are

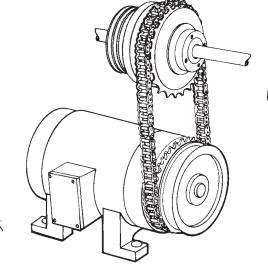
equipped with flanges designed to bolt

directly to a universal joint assembly.

Drive shaft alignment must be

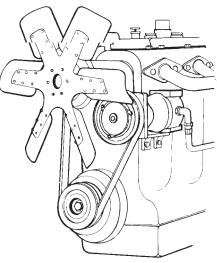
shaft with a universal joint or other flexible coupling is attached to the

sor on the other. The clutch usually



Parallel Shaft

In parallel shaft applications, torque is transmitted from a drive or line shaft to the pump, compressor or other accessory which is mounted parallel to it. The straight bore clutches with bearing mounted fields are often used in this drive arrangement. Belt or chain drives are most frequently used in parallel shaft applications.



Belt Drive

The most common method of driving mobile accessories is through automotive or industrial belts. The driven accessory is parallel to and driven by a pulley mounted on the engine crankshaft. The relative diameters of the drive and driven pulleys, speed range of the engine, and required pump or compressor performance are all factors to be considered when selecting a belt driven clutch. Warner Electric clutches have AB grooved pulleys, which permit the use of either an A or B V-belt.

within 3°.

Direct Drive

Electrical

Electrical Ratings

All current and resistance ratings are taken at ambient temperatures of $70^{\circ}F$ (20°C).

Voltage Requirements

Warner Electric clutches are normally furnished with 12 VDC coils. Clutches can be designed to accommodate other voltages.

Current Draw

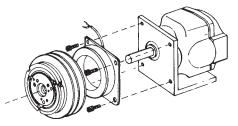
Current draw for each clutch model is listed in the product specifications section of this section.

Resistance/Heat Dissipation

Electrical resistance increases with coil temperature. Since the increase in electrical resistance reduces coil current, the torque transmitted by the clutch will be reduced. In applications where heat dissipation from the clutch is not adequate, air from an external source should be forced over the clutch to ensure proper operation. Most Warner Electric clutches shown in this section have been designed to operate in typical under hood temperatures.

Clutch Mounting

Warner Electric clutches are offered with flange or bearing mounts. Select the type best suited for your application based on the information below.



Bearing Mounted Clutches

Bearing mounted clutches are pre-assembled into a complete operating unit which is mounted directly to the shaft. In this design, the field is mounted on its own bearing as an internal part of the clutch and has an antirotation tang to prevent it from turning in operation. This antirotation tang is to be pinned LOOSELY to a member or held with a torque arm.

Flange Mounted Clutches

In a flange mount clutch application, the field is bolted directly to a fixed member on either the output (engine crankcase or electric motor) or the driven accessory (input). Mounting brackets and fixtures for a specific application must be designed in accordance with the clutch dimensions found in the specifications section to ensure proper perpendicularity and concentricity.

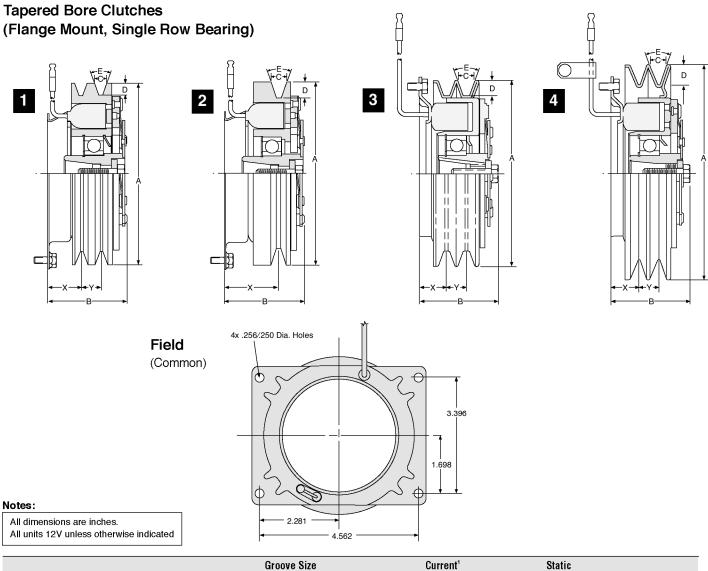
Clutch Location

Wherever possible, the clutch should be located on the higher speed shaft.

Clutch Rotation

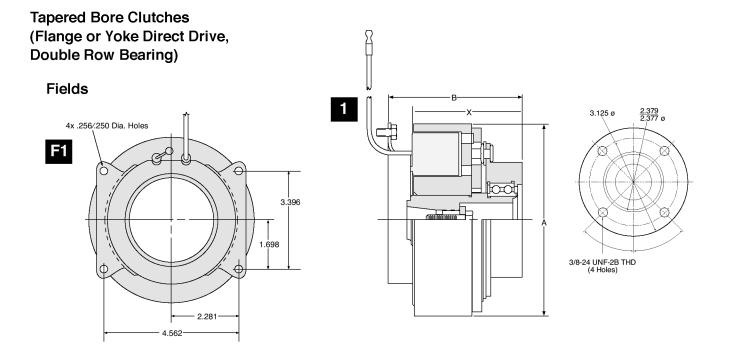
Direction of drive can be a significant design consideration in applications with a peak load during clutch engagement. Warner Electric clutches incorporate leaf springs in the armature to transmit the load. When peak loads at start-up are possible, springs should be oriented so that they are placed in tension (stretch).

Clutch rotation can be determined by observing the leaf spring direction on the armature.



					(Groove Siz	e	-	Current ¹		Static		
A	Clutch Type	В	X	Y	C	D	E	Gage Line	Draw Amps	Resistance Ohms ¹	Torque Ib ft	Rotation	Model No.
5.75	1	2.66	1.10	.59	.38	.36	36°	.841	4.55	2.64	75	CW	1436-97
5.90	1	2.66	1.17	.64	.38	.36	36°	.841	4.55	2.64	75	CW	1436-18
6.00	2	2.29	1.50	_	.50	.55	36°	.841	4.55	2.64	75	CW	1436-78
6.00	2	2.66	1.81	—	.50	.55	36°	.841	4.55	2.64	75	CW	1436-90
6.00	3	2.44	.89	.62	.50	.42	36°	.841	4.651	2.58	75	CW	1411-18
6.00	3	2.44	.89	.62	.50	.42	36°	.841	2.59	9.28	75	CW	1411-35 ²
6.25	2	2.66	2.13	—	.50	.50	36°	.841	4.55	2.64	75	CW	1406-70
6.70	2	2.82	2.38	_	.50	.50	36°	.841	4.55	2.64	75	CW	1436-19
7.00	4	2.50	.89	.62	.50	.55	36°	.841	4.651	2.58	75	CW	1411-72

¹Cold current draw ²24V



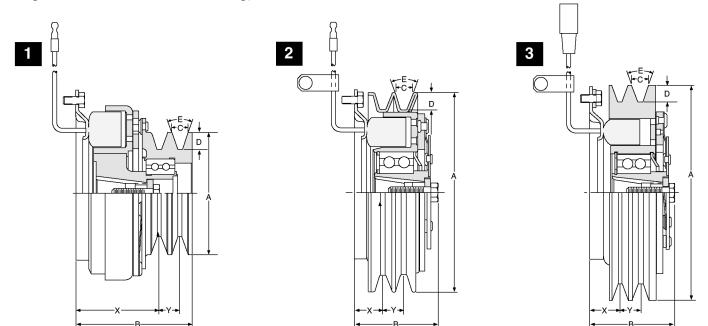
Notes:

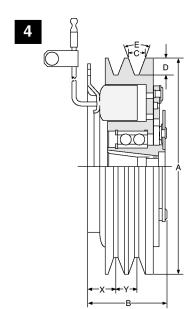
All dimensions are inches. All units 12V unless otherwise indicated Drive shaft alignment must be within 3° maximum. (Clutch centerline must be within 3° of power source centerline).

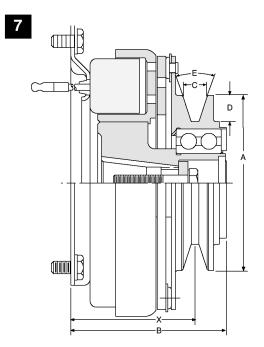
	Clutch	Field				(Groove Siz	e	0	Current ¹	Desistance	Static		Madal
A	Type	Field Type	В	X	Y	C	D	E	Gage Line	Draw Amps	Resistance Ohms ¹	Torque Ib ft	Rotation	Model No.
6.14	1	F1	4.42	3.72	_	_	_	_	.841	4.58	2.62	200	CCW	1415-1

¹Cold current draw

Tapered Bore Clutches (Flange Mount, Double Row Bearing)

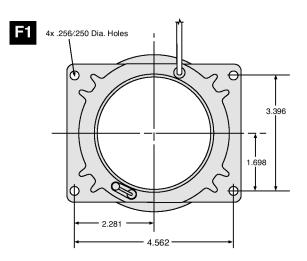






Tapered Bore Clutches (Flange Mount, Double Row Bearing)

Fields



Notes:

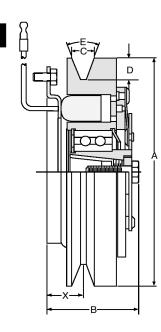
All dimensions are inches. All units 12V unless otherwise indicated

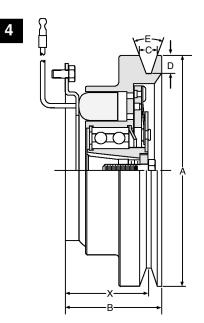
	0 Just a la	F 1.14				(Groove Siz	e	0	Current ¹	Desistence	Static		M
A	Clutch Type	Field Type	В	X	Y	C	D	E	Gage Line	Draw Amps	Resistance Ohms ¹	Torque Ib ft	Rotation	Model No.
3.52	7	F1	3.122	2.5	—	—	.52	38°	.841	4.99	2.41	75	CW	1411-96
3.75	1	F1	3.62	2.55	.62	.50	.53	36°	.841	4.99	2.406	75	CW	1411-69
6.00	2	F1	2.50	.91	.62	.50	.42	36°	.841	4.651	2.58	75	CW	1411-39
6.00	2	F1	2.50	.91	.62	.50	.42	36°	.841	2.59	9.28	75	CW	1411-50 ²
6.00	2	F1	2.51	.91	.62	.50	.42	36°	.841	4.36	2.752	90	CW	1411-67
6.00	2	F1	2.51	.91	.62	.50	.42	36°	.841	4.36	2.752	90	CW	1411-55 ³
6.00	2	F1	2.46	.91	.62	.50	.42	36°	.841	4.651	2.58	75	CW	1411-68 ³
6.00	2	F1	2.54	.91	.62	.50	.42	36°	.841	4.651	2.58	75	CW	1411-54 ³
6.00	3	F1	2.66	1.32	.62	.50	.55	36°	.841	4.55	2.64	75	CW	1466-21
6.70	4	F1	2.57	.88	.62	.50	.55	36°	.841	4.952	2.423	120	CW	1466-53
7.00	2	F1	2.50	.91	.62	.50	.56	36°	.841	4.651	2.58	75	CW	1411-41
7.00	2	F1	2.50	.91	.62	.50	.56	36°	.841	4.36	2.752	90	CW	1411-42
7.00	2	F1	2.51	.91	.62	.50	.55	36°	.841	4.36	2.752	90	CW	1411-56 ³
7.00	2	F1	2.50	.91	.62	.50	.56	36°	.841	2.59	9.28	75	CW	1411-86 ²

¹Cold current draw ²24V ³Special terminal

2

Tapered Bore Clutches (Flange Mount, Double Row Bearing)





Same Field as page 11

Notes:

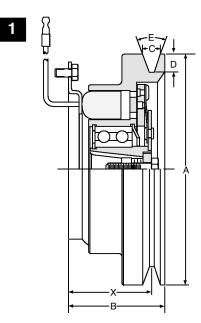
All dimensions are inches. All units 12V unless otherwise indicated

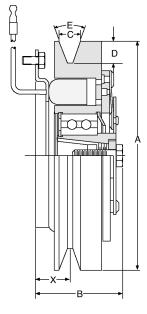
					0	Groove Siz	e		Current ¹	D · · ·	Static		
A	Clutch Type	В	X	Y	C	D	E	Gage Line	Draw Amps	Resistance Ohms ¹	Torque lb ft	Rotation	Model No.
6.00	2	2.48	.93		.60	.52	38°	.841	4.55	2.64	75	CW	1466-69
6.00	2	2.48	1.50		.60	.52	38°	.841	4.55	2.64	75	CW	1466-70
6.00	2	2.66	1.41	_	.50	.55	36°	.841	4.55	2.64	75	CW	1466-20
6.00	2	2.66	1.24	_	.66	.59	38°	.841	4.55	2.64	75	CW	1466-26
6.10	2	2.78	1.35	_	.66	.55	38°	.841	2.54	9.43	75	CW	1466-64 ²
6.70	4	2.79	2.38		.50	.50	36°	.841	4.55	2.64	75	CW	1406-19
6.70	4	2.79	2.38	_	.50	.50	36°	.841	2.545	9.43	75	CW	1436-41 ²
6.70	4	2.79	2.05		.50	.50	36°	.841	2.545	9.43	75	CW	1436-48 ²
6.70	4	2.79	2.05		.60	.56	38°	.841	4.55	2.64	75	CW	1406-43

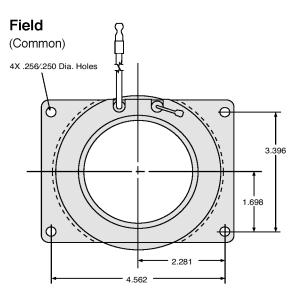
¹Cold current draw ²24V

Tapered Bore Clutches (Flange Mount, Double Row Bearing)

2







Notes:

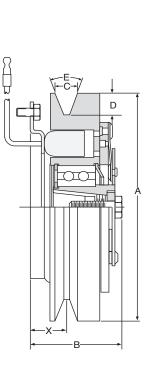
All dimensions are inches.

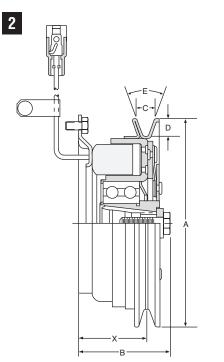
All units 12V unless otherwise indicated

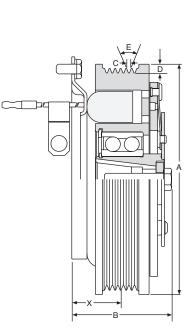
					G	Groove Siz	e	-	Current ¹		Static		
A	Clutch Type	В	X	Y	C	D	E	Gage Line	Draw Amps	Resistance Ohms ¹	Torque Ib ft	Rotation	Model No.
6.70	1	2.84	2.38	—	.60	.56	38°	.841	4.55	2.64	75	CW	1406-44
6.70	1	2.53	1.95		.60	.56	38°	.841	4.55	2.64	75	CW	1406-49
6.70	1	2.53	1.95	_	.60	.56	38°	.841	2.545	9.43	75	CW	1436-55 ²
6.70	1	2.84	2.37	_	.66	.56	38°	.841	4.55	2.64	75	CW	1406-47
6.70	2	2.48	1.02	_	.60	.56	38°	.841	4.55	2.64	75	CW	1406-34
6.70	2	2.48	1.02	_	.60	.56	38°	.841	2.545	9.43	75	CW	1436-49 ²
6.70	2	2.48	1.18		.60	.56	38°	.841	4.55	2.64	75	CW	1406-42
6.70	2	2.48	1.18	_	.60	.56	38°	.841	2.545	9.43	75	CW	1436-51 ²

¹Cold current draw ²24V

Tapered Bore Clutches (Flange Mount, Double Row Bearing)







4

Same Field as page 11

Notes:

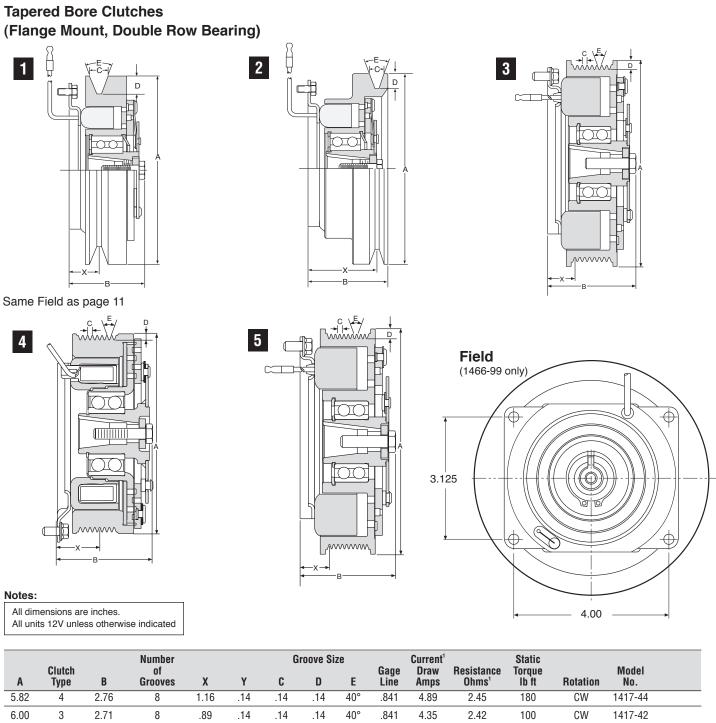
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All dimensions are inches. All units 12V unless otherwise indicated

	Clutch		Number			0	Groove Siz	e	Como	Current ¹	Resistance	Static		Model
A	Туре	В	of Grooves	Х	Y	C	D	Е	Gage Line	Draw Amps	Ohms ¹	Torque Ib ft	Rotation	No.
5.79	4	2.53	6	1.24	.14	.14	.14	40°	.841	2.5	9.61	75	CW	1466-105 ²
6.70	1	2.48	1	1.02	—	.50	.50	36°	.841	4.55	2.64	75	CW	1406-39
6.70	1	2.48	1	1.02	—	.50	.50	36°	.841	2.50	9.61	75	CW	1466-68 ^{2,3}
6.70	1	2.48	1	.87	—	.50	.50	36°	.841	2.545	9.43	75	CW	1466-84 ²
6.70	2	2.97	1	2.29	_	.78	.68	38°	.841	4.651	2.58	75	CW	1411-49
6.70	2	2.73	1	2.05	_	.79	.69	38°	.841	4.651	2.58	75	CW	1411-61
6.70	2	2.52	1	1.95	—	.50	.50	36°	.841	4.651	2.58	75	CW	1411-70

¹Cold current draw ²24V

³Special terminal

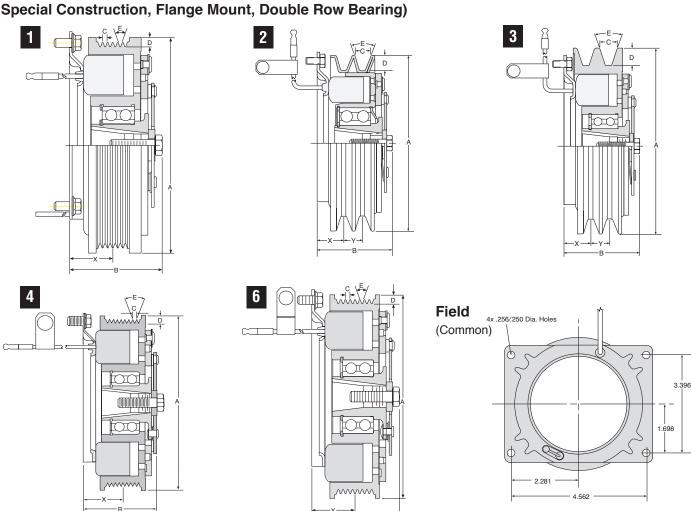


	Cluster		Number			G	roove Si	ze	Corro	Current ¹	Desistance	Static		Model
А	Clutch Type	В	of Grooves	Х	Y	C	D	Е	Gage Line	Draw Amps	Resistance Ohms ¹	Torque Ib ft	Rotation	No.
5.82	4	2.76	8	1.16	.14	.14	.14	40°	.841	4.89	2.45	180	CW	1417-44
6.00	3	2.71	8	.89	.14	.14	.14	40°	.841	4.35	2.42	100	CW	1417-42
6.00	3	2.71	8	.89	.14	.14	.13	40°	.841	2.47	9.73	100	CW	1466-99 ²
6.00	3	2.71	8	.89	.14	.14	.13	40°	.841	4.35	2.42	100	CW	1473-69
6.00	5	2.71	10	.85	.14	.14	.14	40°	.841	4.35	2.42	100	CW	1417-49
6.92	1	2.48	1	1.02	_	.80	.69	38°	.841	4.55	2.64	75	CW	1406-41
7.00	1	2.48	1	1.19	_	.79	.69	38°	.841	4.55	2.64	75	CW	1406-97
7.25	2	2.82	1	2.29		.79	.88	38°	.841	4.55	2.64	75	CW	1466-92 ⁴

¹Cold current draw ²24V

^₄A/B groove

Engine, Pump and Compressor Clutches—Dimensions



Tapered Bore Clutches (Special Construction, Flange Mount, Double Row Bearing)

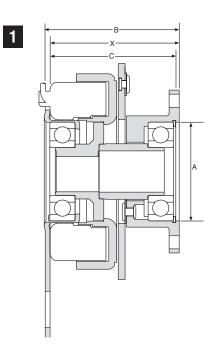
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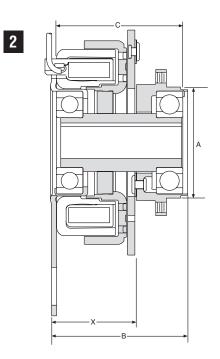
Clutches on this page are specially constructed to withstand more severe operating environments. All dimensions are inches. All units 12V unless otherwise indi-

	Clutch		Number of			(Groove Siz	e	Gage	Current ¹ Draw	Resistance	Static Torque		Model
A	Туре	В	Grooves	х	Y	C	D	Е	Line	Amps	Ohms ¹	lb ft	Rotation	No.
5.75	1	2.57	6	1.20		.14	.14	40°	.841	4.952	2.423	120	CW	1417-11
5.80	6	2.61	6	1.20	.14	.14	.14	40°	.841	2.47	9.73	100	CW	1417-41
6.00	2	2.52	2	.91	.62	.50	.42	36°	.75	4.36	2.752	90	CW	1417-2 ²
6.00	2	2.57	2	.91	.62	.50	.42	36°	.841	4.36	2.752	90	CW	1417-8
6.00	4	2.60	7	.85	.14	.14	.14	40°	.84	4.95	2.42	120	CW	1417-18
6.22	3	2.46	2	.85	.73	.61	.64	34°	.841	4.36	2.752	90	CW	1417-9
6.70	3	2.57	2	.88	.62	.50	.55	36°	.841	4.952	2.423	120	CW	1417-4
6.89	3	2.57	2	.78	.75	.61	.63	34°	.841	4.952	2.423	120	CW	1417-12
7.00	2	2.52	2	.91	.62	.50	.55	36°	.75	4.36	2.752	90	CW	1417-1 ²
7.00	2	2.57	2	.91	.62	.50	.56	36°	.841	4.36	2.752	90	CW	1417-7

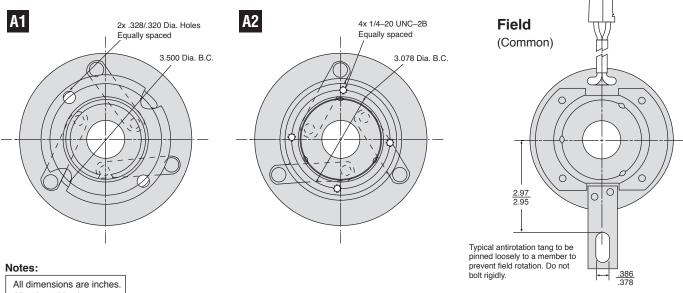
¹Cold current draw ²8:1 Taper ³A/B grooves ⁴24V

Straight Bore Clutches (Bearing Mount)





Armatures



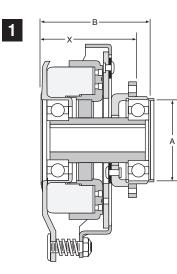
All dimensions are inches. All units 12V unless otherwise indicated

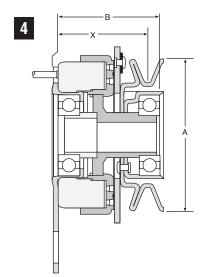
Bore Size	Clutch Type	Armature Type	A	В	х	C	Static Torque Ib ft	Current ¹ Draw Amps	Resistance Ohms ¹	Rotation	Keyway	Model No.
3/4"	2	A2	2.64	3.31	2.60	2.635	80	.56	161	CCW	.189	5215-105 ²
1"	1	A1	2.44	3.375	3.21	3.045	70	4.89	2.45	CCW	.25	5215-60

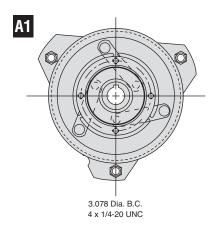
¹Cold current draw ²90 Volts

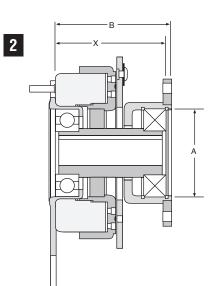
Engine, Pump and Compressor Clutches—Dimensions

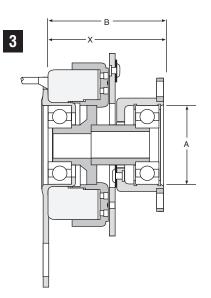
Straight Bore Clutches (Bearing Mount)

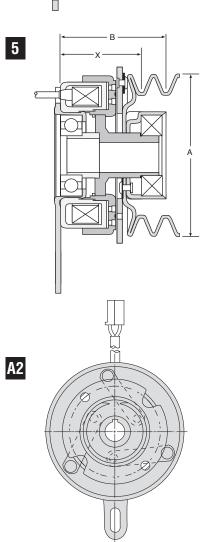






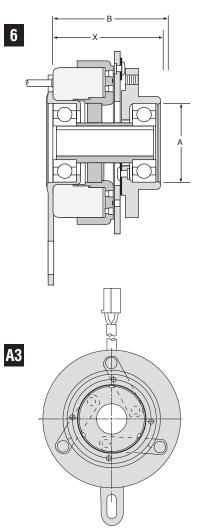






3.50 Dia. B.C.

2 x .324

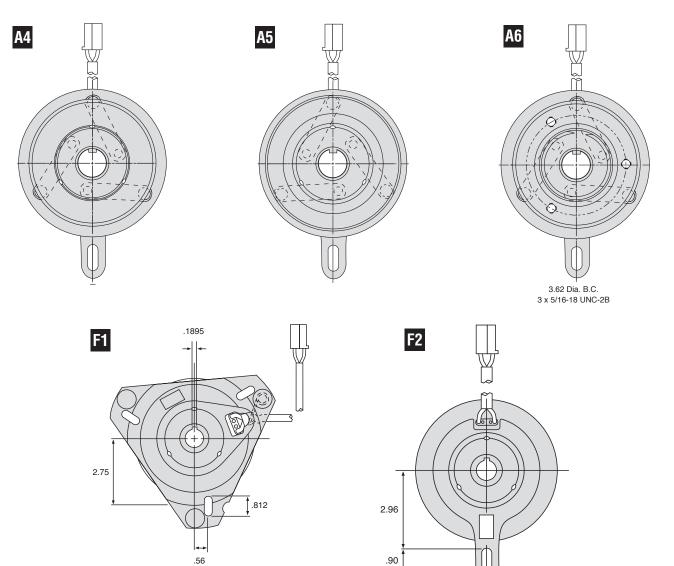


3.50 Dia. B.C. 4 x .284

16

.382

Straight Bore Clutches (Bearing Mount)



Notes:

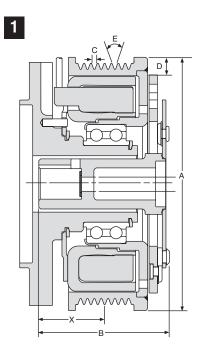
All dimensions are inches. All units 12V unless otherwise indicated

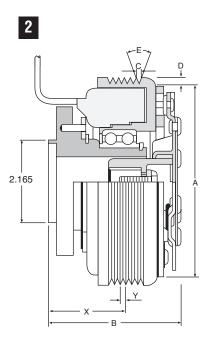
Bore	Clutch	Armature	Field		Pit	ch diamo	eter						Static	Current ¹ Draw	Resistance			Part
Size	Туре	Туре	Туре	A	A Belt	B Belt	В	Х	Y	C	D	Е	Torque Ib ft	Amps	Ohms ¹	Rotation	Keyway	No.
3/4"	1	A1	F1	2.64	—	—	3.41	2.95	—	—	—	—	105	4.51	2.66	CCW	.19	5215-66
3/4"	2	A2	F2	2.44	—	—	3.40	3.21	—	—		—	70	4.51	2.66	CCW	.19	5215-67
1"	3	A2	F2	2.44	—	—	3.21	3.21	_	—	_	—	105	4.84	2.48	CCW	.25	5215-60
1"	3	A3	F2	2.64	—	—	2.96	2.60	_	—	_	—	105	4.84	2.48	CCW	.25	5215-57
1"	4	A4	F2	4.6	—	4.62	3.52	2.77	_	.63	.55	36°	70	4.51	2.66	CCW	.25	5215-63
1"	5	A5	F2	5.1	5.07		3.73	2.37	.67	.49	.48	34°	70	4.51	2.66	CW	.25	5215-77
1"	6	A6	F2	3.00	_	—	3.08	2.30	_	—	—	_	105	4.84	2.48	CCW	.25	5215-82

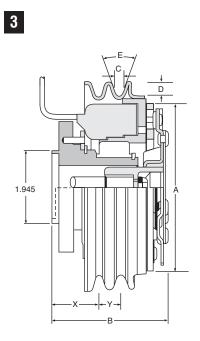
¹Cold current draw

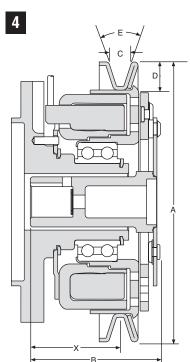
Straight Bore Clutches (Special Construction, Flange Mount)

(The clutches on this page include mounting hub)

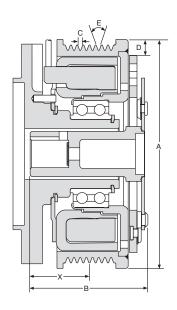




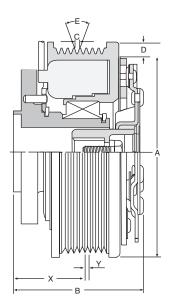




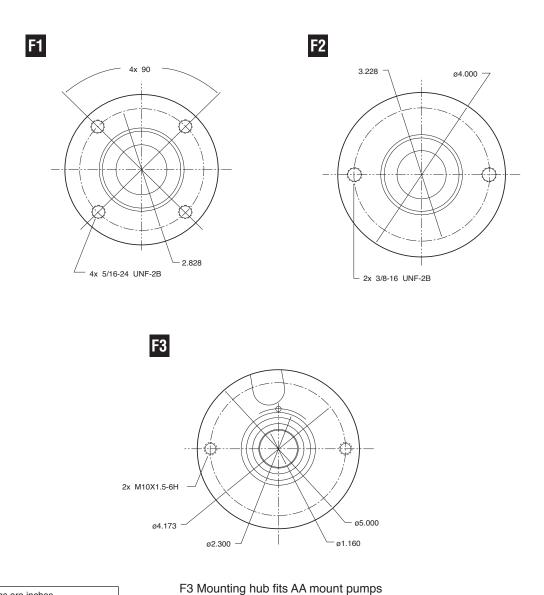
5



6



Mounting Hubs

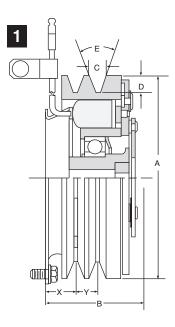


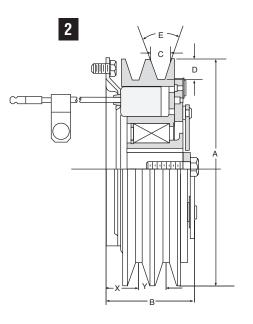
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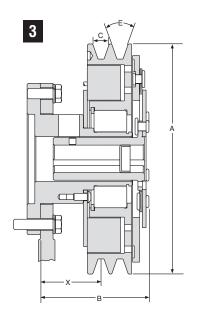
All dimensions are inches. All units 12V unless otherwise indicated

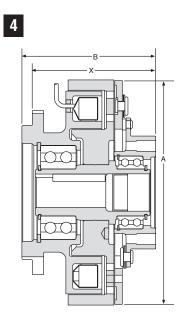
Mounting Static Current¹ Clutch Draw Resistance Model Bore Hub Torque Rotation Size Туре Type A В Х Υ C D Ε lb ft Amps Ohms¹ Keyway No. 4.91 3.19 1.24 .14 .14 40° 4.92 CW 1417-24 1/2' 2 F2 .13 90 2.44 .16 1/2" 3 F1 5.59 3.19 1.60 .56 .38 .36 36° 90 4.92 2.44 CW .16 1417-26 .532 6 F2 4.96 2.95 1.44 .14 .14 .14 40° 70 4.63 2.59 CW .16 1417-31 CCW .627 4 F3 5.76 2.83 2.23 ____ 0.437 0.42 36° 90 2.523 9.513 0.158 1417-34² .627 4 F3 5.76 2.83 2.23 0.437 36° 2.523 9.513 CW 1417-35² _ 0.42 90 0.158 5 .627 F3 5.00 2.83 1.54 0.14 0.14 0.14 40° 90 2.523 9.513 CW 0.158 1417-45² .627 F3 2.83 1.54 40° 90 2.52 9.51 CW 0.158 1417-48 1 5.00 0.14 0.14 0.14

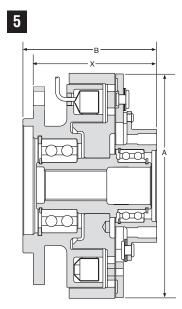
Straight Bore Clutches (Special Construction)



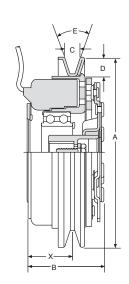






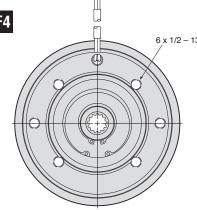


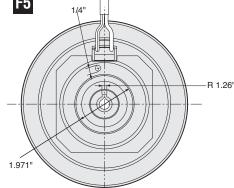


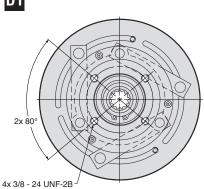


F1 F2 F3 П-4x .256/.250 Dia. Holes 14.312 F 0 \odot Ο C 3.396 1.698 6X 60fl NOM 0 0 4 2.281 4.562 F4 F5 D1 1/4" 6 x 1/2 – 13 R 1.26"

Fields







6X Ø .437"

-Ø 4.750

Notes:

All dimensions are inches.
All units 12V unless otherwise indicated

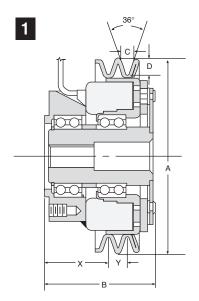
	0	F 1.14	D. i			Pitch	diamete	r					Static	Current ¹	Desistance			M - 4 - 1
Bore Size	Clutch Type	Field Type	Drive Coupling	A	A Belt	B Belt	В	Х	Y	C	D	Е	Torque Ib ft	Draw Amps	Resistance Ohms ¹	Rotation	Keyway	Model No.
9/16"	1	F1	_	5.98	5.73		2.63	.89	63	.51	.46	38°	75	4.55	2.65	CW	.127	1417-16
3/4"	2	F1	—	6.22	_	5.88	2.56	.85	.73	.61	.63	34°	90	4.36	2.752	CW	.189	1417-10
3/4"	2	F1	_	6.22	_	5.88	2.56	.85	.73	.61	.63	34°	90	2.59	9.28	CW	.189	1417-13 ²
1"	3	F2	_	8.36	_	8.00	3.98	2.19	.69	.60	.54	38°	200	5.538	2.167	CW	.252	1415-4 ³
1"	4	F3	D1	7.56	—	—	4.469	4.11	_	_	_		200	5.54	2.17	CCW	0.25	1415-5
Sp	5	F4	D1	7.56	_	_	4.469	4.11	_	_	_	_	200	5.54	2.17	CCW	_	1415-7 ⁴
1"	4	F3	D1	7.56	_	_	4.469	4.11	_	_	_	_	200	3.35	7.16	CCW	0.25	1415-8 ²
1"	3	F2	D1	8.36	_	8.00	3.98	2.19	.69	.60	.54	38°	200	3.35	7.16	CW	0.252	1415-10 ^{2, 3}
.532"	6	F5	_	5.63	.44		2.43	1.37	_	.44	.42	36°	75	3.38	3.55	CW	0.16	1417-43

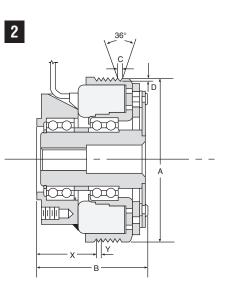
¹Cold current draw ²24V ³For Gresen pump only

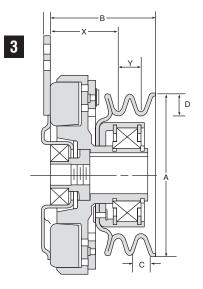
⁴ 13 tooth spline hub clutches must be installed on pumps with internal involute flat root side fit splines per ANSI-892. Major diameter .901" max.

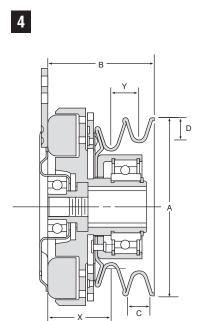
Engine, Pump and Compressor Clutches—Dimensions

Straight Bore Clutches (Shaft Mount)

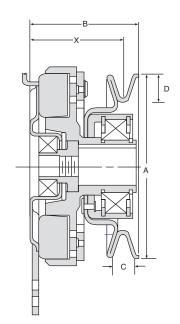




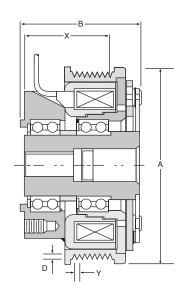


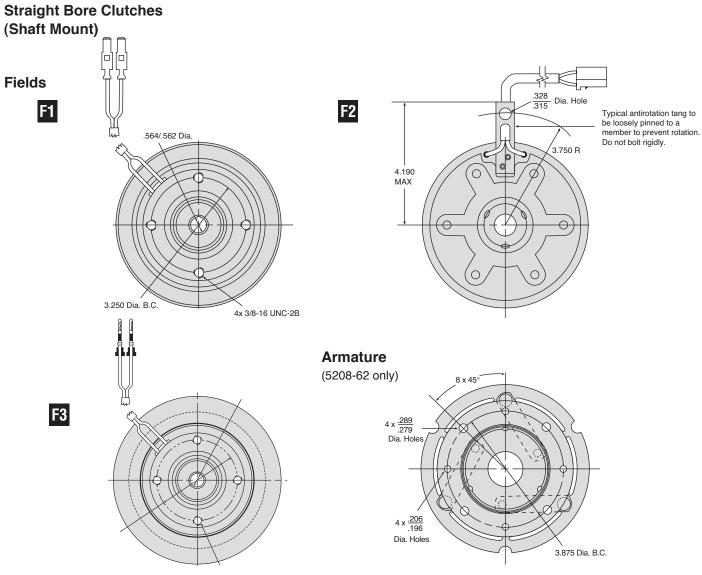


6



7





Engine, Pump and Compressor Clutches—Dimensions

Notes:

All dimensions are inches. All units 12V unless otherwise indicated

Bore	Clutch	Field		Pitch d	iameter						Static Torque	Current ¹ Draw	Resistance			Model
Size	Туре	Туре	Α	A Belt	B Belt	В	Х	Y	C	D	Ib ft	Amps	Ohms ¹	Rotation	Keyway	No.
9/16"	1	F1	5.65			3.30	1.88	.56	.380	.38	75	3.41	3.52	CW	.127	1417-14 ²
9/16"	2	F1	5.00		_	3.30	1.74	.14	.120	.126	75	3.41	3.52	CW	.127	1417-15 ²
9/16"	1	F1	6.05	_	_	3.30	1.752	.65	.44	.46	75	3.41	3.52	CW	.127	1417-17
9/16"	1	F1	6.05	_	_	3.30	1.752	.65	.44	.46	75	2.53	9.51	CW	.127	1417-38 ³
9/16"	7	F1	6.00	_	_	3.34	2.308	.14	.120	.13	75	3.38	3.52	CW	.13	1417-28
1"	3	F2	4.93	4.50	_	3.48	2.027	.67	.49	.48	70	4.17	2.88	CCW	.25	5208-55
1"	4	F2	5.22	_	^{3.98} / _{4.51}	3.53	1.859	.82	.637	.55	70	4.17	2.88	CCW	.25	5208-4
1 ¹ / ₈ "	3	F2	4.93	4.50	_	3.48	2.027	.67	.494	.48	70	4.17	2.88	CCW	.25	5208-29
1 ¹ / ₈ "	4	F2	5.22	_	^{3.98} / _{4.51}	3.53	1.859	.82	.637	.55	70	4.17	2.88	CCW	.25	5208-3
1 ¹ / ₈ "	6	F2	5.36	4.74	5.08	3.34	2.52	_	.612	.632	70	4.17	2.88	CCW	.25	5208-40
			0				-									

¹Cold current draw ²Bracket mounting ³24 Volt For all models shown: E Nom. 1.18, F Max. .170, G Max. 1.20, H Max. 1.47.

Acceleration Time, Engagement Time

The time required to change the speed of a system from the moment the clutch receives the appropriate electrical signal until the clutch is fully engaged and the system is moving at its maximum speed.

Bearing Mount

A clutch which is preassembled into a complete operating unit and is mounted directly to the shaft.

Brushholder

A clutch component which carries electrical current from the lead wires to the rotating magnet.

Build Up Time

The time in seconds required to build up to 90% of rated flux which corresponds to 80% rated torque.

Burnishing

The process of cycling or "wearing in" of clutch or brake friction surfaces. This process ensures rated torque during initial cycles, and decreases the cycles required from installation to full rated torque output.

Decay Time

The time in seconds required to decay to 10% of rated flux which corresponds to 1% of rated torque on de-energization of the unit.

Deceleration Time, Engagement Time

The time required to stop a system from the moment the brake receives the appropriate electrical signal until the brake is statically engaged and the system is at rest.

Field

A component part of Warner Electric clutches consisting of a steel shell and a coil. Also referred to as a magnet.

Flange Mount

A clutch which has the field bolted directly to a fixed member on the machine.

Flux

Magnetic attraction caused by an electrical current.

Gap

The distance between armature and rotor faces in clutches when the unit is in an inactive state (i.e. disengaged).

Integral Key

A key shaped directly into the bore of a clutch. This is sometimes used in place of a standard keyway and key.

K Factor

See service factor.

Poles

1. Refers to magnet poles: North/South poles. 2. The edges of a Warner Electric magnet or field shell through which the magnetic flux flows.

Pulley

A sheave that turns or is turned by a belt so as to transmit torque, rotation.

Residual Magnetism

The condition in magnets where low level magnetism remains after the electric current is removed.

Rotor

The input member of a clutch/brake.

Service Factor

A figure by which torque is multiplied to ensure performance of the clutch under the worst case application conditions.

Tapered Bore (Shaft)

Many hydraulic pumps incorporate a taper on the output shaft, providing stronger clutch-to-shaft engagement than on straight shafts. Tapered shafts are most commonly in 4:1 and 8:1 taper ratios.

4:1 Taper: The shaft changes in diameter by one inch for each four inches of length.

8:1 Taper: The shaft changes in diameter by one inch for each eight inches of length.

Torque

Static: The torque which is developed when there is no relative motion or slippage between the mating friction surfaces. A clutch which is fully engaged and driving exhibits static torque. All standard units are rated on the basis of static torque after burnishing.

Dynamic: The torque developed when there is relative motion between the mating friction surface. The torque varies inversely with the amount of slip, so specific values must be taken from engineering data.

Clutch Location

Wherever possible, the clutch should be located on the higher speed shaft.

Clutch Rotation

Direction of drive can be a significant design consideration in applications with a peak load during clutch engagement. Warner Electric clutches incorporate leaf springs in the armature to transmit the load. Where peak loads at start-up are possible, springs should be oriented so that they are placed in tension (or stretch).

Spring rotation can be determined by observing the leaf spring direction on the armature.

Electrical Ratings

All current and resistance ratings are taken at ambient temperatures of $70^{\circ}F$ (20°C).

Fluid Power Formulae

If you are sizing a clutch for a pump application, but do not know the HP required, the following formula will allow you to work back to the torque formulae.

If HP is unknown:

HP = ______

Pump Efficiency

Where:

- GPM = Fluid flow in gallons per minute
 - PSI = Pressure in pounds per square inch
- Pump Efficiency = normally 85%

Rules of Thumb:

1 HP per gallon @ 1500 psi

.7 HP per gallon@ 1000 psi

If PSI is unknown:

1 cubic inch per revolution equals 16 lb.in. of torque per 100 psi.

1 gallon equals 231 cubic inches.

If GPM is unknown:

 $GPM = \frac{RPM \times DISP (IN^3)}{231}$

Static Torque

The torque requirements for your particular application may be determined by using the following relationship:

$$T = \frac{5250 \times HP}{RPM}$$
$$T = \frac{CIR \times PSI}{75.4}$$

Voltage Requirements

Most clutches and clutch/brakes require 12 VDC to operate at their maximum torque rating. Less than 12 VDC may cause clutch slippage and premature failure.

Abbreviations:

T = Torque

HP = Horsepower

RPM = Speed of clutch (Revolutions per minute)

CIR = Cubic inch per revolution

PSI = Pounds per square inch

K = Service factor

Part Number Index

1408-34 11 10882 502730 1417-13 21 5208-40 23 1406-39 12 10681 502730 1417-15 21 13279 5208-60 23 5215-60 15 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17 5215-60 17	Warner	Page	Pitts	Ogura	Warner	Page	Pitts	Ogura	Warner	Page	Pitts	Ogura
1406-39 12 10631 502730 1417-14 23 13279 - 5215-65 12 - - 1406-41 13 10533 502744 1417-16 23 13829 - - 5215-60 17 - - 1406-42 11 10694 502732 1417-17 23 - - 5215-60 17 - - 1406-43 10 10591 502730 1417-28 19 - - 5215-66 17 - - 1406-49 11 10565 502741 1417-28 19 - - 5215-77 17 - - - 5215-77 17 - - - 1417-44 19 - - 5215-82 17 - - - - 1417-44 13 - - - 17 - - - 1417-44 13 - - - 1417-44 13 - - - 1417-44 13 - - - - <	1406-19	10	10595	502740	1417-12	14	_	_	5208-29	23	_	_
1406-41 1071-5 23 13829 - 5216-60 15 - - 1406-42 11 10047 502732 1417-16 21 - - 5216-60 17 - - 1406-43 10 10391 502732 1417-18 14 - - 5216-60 17 - - 1406-44 11 10696 502739 1417-28 19 - - 5216-60 17 - - 1406-45 11 10695 502741 1417-28 23 - - 5216-60 17 - - 1406-47 11 10695 502743 1417-31 19 - - 5216-60 17 - - 1406-70 13 10761 50200 1417-38 23 - - - 5216-50 15 - - 1406-77 13 10761 502040 1417-41 14 - - - 5216-50 15 5 - - 1411-18 0 1224 1417-41 13 - - - - - - - - - -	1406-34	11	10582	502731	1417-13	21	_	_	5208-40	23	_	-
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For Application Assistance, Phone 815-389-6369 or Fax 815-389-7648

Date	
Company Name	
Address	
City	
Type of Application	Electrical System Regulated Unregulated
Shaft Diameter (give limits) Straight	Clutch Duty Cycle (Time On/Off)/Hr Environment Temp Range °F Location Quantity Annual
Power Source (give HP and Mfg) Gas Diesel Electric Other Driven Load (give parameters)	
Air Compressor Pump Mower Deck	

Note: Refer to the individual sections of this catalog for details.

Altra Industrial Motion

Warner Electric

Electromagnetic Clutches and Brakes - USA

South Beloit, IL 815-389-3771

For customer service: 1-800-825-6544 For application assistance: 1-800-825-9050

Electromagnetic Clutches and Brakes - Europe

St Barthelemy d'Anjou, France +33 (0)2 41 21 24 24

For sales office +33 (0)2 41 21 24 76

Precision Electric Coils and Electromagnetic Clutches and Brakes - ŬSA Columbia City, IN 260-244-6183

Inertia Dynamics

Spring Set Brakes; Power On and Wrap Spring Clutch/Brakes New Hartford, CT 860-482-4444

Matrix International

Electromagnetic Clutches and Brakes, Pressure Operated Clutches and Brakes

Brechin, Scotland +44 (0) 1356 602000

South Beloit, IL 815-389-3771

Warner Linear

Linear Actuators and Guideways - USA Belvidere, IL

815-547-1106

For application assistance: 1-800-825-9050

TB Wood's

Belted Drives and Flexible Couplings

Chambersburg, PA 717-264-7161 For assistance: 1-888-829-6637

Press #5 – Customer Service Press #7 – Mechanical Applications

Wichita Clutch and Industrial Clutch

Pneumatic and Oil Immersed Clutches and Brakes - USA

Wichita Falls, TX 940-723-3400

Pneumatic Clutches and Brakes - Europe Bedford, England +44 (0)1234 350311

Twiflex Limited

Caliper Brakes and Thrusters Twickenham, England +44 (0) 20 8894 1161

Formsprag Clutch

Overrunning Clutches and Holdbacks Warren, MI 586-758-5000

For application assistance: 1-800-927-3262

Marland Clutch

Roller Ramp and Sprag Type Overrunning Clutches and Backstops Burr Ridge, IL

630-455-1752 Stieber Clutch

Overrunning Clutches

and Holdbacks Heidelberg, Germany +49 (0)6221 30 47 0

Boston Gear

Enclosed and Open Gearing, Electrical and Mechanical P.T. Components Charlotte, NC 704-688-7300

For customer service: 1-800-825-6544 For application assistance: 1-800-816-5608

Huco Dynatork

Precision Couplings and Air Motors Hertford, England

+44 (0) 1992 501900 US

800-825-6544

Erie, PA

Ameridrives Couplings

Gear Couplings, Mill Spindles, Universal Joints

814-480-5000 Universal Joints, Drive Shafts, Mill Gear Couplings

Green Bay, WI 920-593-2444

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Disc, Gear, Grid Couplings, Overload Clutches Dewsbury, England +44 (0) 1924 460801

Nuttall Gear and Delroyd Worm Gear

Worm Gear and Helical Speed Reducers Niagara Falls, NY 716-298-4100

Saftek Friction

Non-asbestos Brake and Clutch Materials Telford, England +44 (0) 1952 581122

Altra Industrial Motion -Asia Pacific and Africa

China	852 2615 9313
Taiwan	886 2 2577 8156
Singapore	65 6487 4464
Thailand	66 2 322 5527
Australia	612 9894 0133
S. Africa	27 11 918 4270

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